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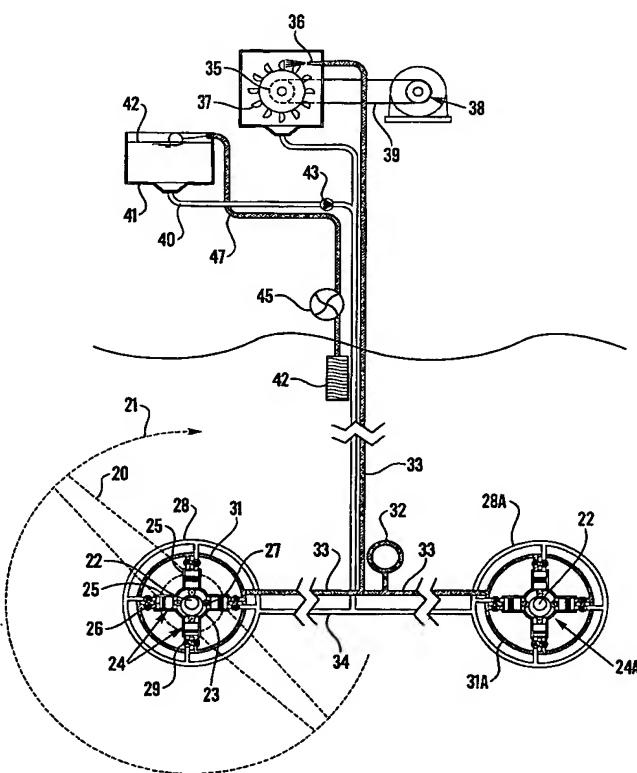
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(54) Title: HYDRAULIC SPEED-INCREASING TRANSMISSION FOR WATER CURRENT POWERED TURBINE



(57) Abstract: A hydraulic transmission system for use with at least one water current driven turbine rotor (6), low speed high pressure pump (24) is arranged to receive operational drive from the turbine rotor, and to deliver its output to a high speed hydraulic motor or turbine (35) in such manner as to produce an increase of the shaft or motor speed accompanied by a corresponding reduction of torque required to drive electrical generators (38) or other high speed machinery.

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HYDRAULIC SPEED-INCREASING TRANSMISSION FOR WATER CURRENT POWERED TURBINE

BACKGROUND OF THE INVENTION

This invention relates to turbines and other such devices capable of interacting with a flow of fluid in such a manner as to transfer energy from the fluid to a mechanical device. This invention relates more particularly to turbines or other such devices arranged to be driven by the action of a flow of water or other liquid medium. Thus, the present invention relates in particular to the use of turbines and similar moving devices for extracting kinetic energy from flowing water for the purposes of utilising such kinetic energy to produce either electricity or shaft power for utilisation for a required purpose. Flowing water (which may be either fresh water or sea water) used in the context of this invention is a characteristic of tidal, marine, estuarial or river currents.

It is known how to use turbines for such purposes. For example, in our British Patents GB 2256011 B, GB 2311566 B and No 2348250 we have disclosed constructions pertaining to water driveable turbines; i.e. rotors supported within the water column of the sea, river or an estuary so that the flow of water may turn the rotor to produce shaft power and hence possibly electricity for utilisation for a required purpose.

- 20 In particular the present invention relates to a power transmission system for use with one or more such turbine rotors for extracting kinetic energy from flowing water and producing therefrom electricity by driving an electrical generator or alternator or for applying the power for some other useful purpose requiring a fast rotating shaft.
- 25 Generally a turbine for extracting kinetic energy from water currents, whether in a river or at sea, includes a rotor capable of interacting with the flow of water in such a way that some of the energy of motion of the passing mass of water causes

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the rotor to rotate. No matter what kind of rotor is used, whether an axial flow or propeller type of rotor or a cross-flow rotor of the so-called Darrieus type, or even some other form of hydrofoil device reacting against the flow of water currents, the rotation of a rotor driven by the movement of water currents will be slow due 5 to the fact that even the fastest of water currents only run at relatively slow speeds in the range from 2 to 4m/s at most and the rotor extremities cannot generally move much faster than 10 to 15m/s. Consequently only small devices can rotate at reasonably high speeds since the larger the device i.e., diameter of the rotor, generally the smaller the angular movement in a given time.

10 Therefore, it becomes necessary to provide some means for speeding up the output gained from the rotation of a slow moving rotor in order to effectively drive an electrical generator or other relatively high speed machinery. For example an axial flow tidal or river current turbine rotor 15m in diameter will typically rotate at about 10 revolutions per minute in a driving water current of 2 to 2.5m/s (4 to 5 knots) but most conventional electricity generators typically 15 require to be driven at shaft speeds of 1000, 1500 or more revolutions per minute. This situation indicates that there is therefore a need to increase the speed of the main turbine shaft by a factor of 100 or, more. The most generally proposed technique, which is prior art, is by using a multistage gearbox. However gearboxes 20 of such a kind are complicated mechanically, heavy to handle, costly, suffer significant wear and require regular maintenance to replace the lubricant conventionally provided between relatively moving components of the turbines. Furthermore, gearboxes conventionally involved also introduce a hazard to the local environment should the lubricating oil leak out for any reason.

25 THE PRIOR ART

It is known how to use turbines for such above mentioned purposes. For example, in our British Patents GB 2256011 B, GB 2311566 B and No 2348250 we have disclosed constructions pertaining to water driveable turbines; i.e. rotors

supported within the water column of the sea, river or an estuary so that the flow of water may turn the rotor to produce shaft power and hence possibly electricity for utilisation for a required purpose.

Thus the concept of immersing a rotor in a water current in order to extract energy
5 has been tested and described, as for example, in our above mentioned British
Patents.

OBJECT OF THE INVENTION

The object of the invention is to provide a method of and means for increasing the effective rotational speed of a rotating member.

10 STATEMENTS OF THE INVENTION

According to a first aspect of the invention there is provided a hydraulic transmission system for use with at least one water current driven turbine rotor, characterised in that a low speed high pressure pump or pumps is/are arranged to receive operational drive from the turbine rotor, and to deliver high pressure
15 output to a high speed hydraulic motor or turbine in such manner as to produce an increase of the shaft or motor speed accompanied by a corresponding reduction of torque required to drive electrical generators or other machinery as compared with the shaft speed of the turbine rotor or rotors.

20 Preferably, a substantially closed hydraulic fluid circuit is provided for said hydraulic fluid associated with the low speed high pressure pump or pumps

Conveniently the hydraulic fluid for said high pressure pump or pumps is/are arranged to be recirculated from a high speed hydraulic motor or turbine through a low pressure fluid return line or lines to feed said low-speed high-pressure hydraulic pump or pumps.

Preferably the hydraulic fluid is water.

Conveniently, any loss of the water from the system is arranged to be made up by drawing replacement water from the water within which said rotor or rotors is/are is operationally located.

5 In a preferred arrangement such make up water is filtered to remove suspended solids or other undesirable pollutants before being stored in a header tank situated in such a position that it can feed make-up water into the system to replace and water lost through leakage.

10 Since the hydraulic fluid is water is drawn from the operating environment of the turbine or turbines, a relatively high degree of fluid leakage can be tolerated, thereby to allow the use of larger than would otherwise be acceptable clearances relative movement for seals, the arrangement being such that the operation of the pumps can be optimised for mechanical efficiency rather than 100% retention of hydraulic fluid.

15 Preferably each said rotor is associated with a plurality of separate pumps arranged to be operationally driven from the associated rotor, and in that the hydraulic fluid inputs to the pumps are connected to receive fluid from a low pressure fluid plenum and the output sides of all said pumps are associated with a common high pressure plenum connecting via high pressure fluid circuit with the 20 hydraulic motor or rotor coupled to drive a generator or other machinery

.. In accordance with a further aspect of the invention a water drivable turbine is provided with a hydraulic power transmission system for increasing the effective speed of the turbine.

In accordance with a further aspect of the invention a hydraulic power transmission system using water as a working fluid is utilised to achieve the desired increment in speed to couple a low speed tidal or river current turbine rotor to a conventional electrical generator or to other high speed machinery.

5 In a preferred arrangement it is proposed to directly couple a low speed, positive displacement hydraulic pump to the turbine rotor drive shaft.

In particular the hydraulic pump may be of various types, such as, for example, a cam or crank driven single or multi-cylinder piston or plunger type of pump, or an internal or external type gear pump, a lobe pump or a vane pump.

10 Conveniently, pumps utilised may be presented in a variety of configurations, such that, for example, a piston or plunger pump may have a multiplicity of cylinders disposed radially around the cam or driving crank on the main shaft, or the cylinders may be disposed axially along a camshaft or crankshaft, or there may be a series of radially disposed cylinders set axially along a camshaft or
15 crankshaft.

The concepts of the invention may be applied to water current turbines with more than one rotor, in which case each rotor will be coupled to a similar low speed, positive displacement pump fed with the same working fluid that the turbine runs in, whereby if the pump is in the sea it will be sea water, if in an inland river it
20 may be freshwater. Therefore any minor leakage of hydraulic fluid will not create any environmental problem of polluting the surroundings and is acceptable.

25 In a particular arrangement individual cylinders (with piston or plunger pumps) or the pump output will draw in water via non-return valves from a low pressure or supply manifold and similarly they will deliver at a higher pressure pumped water through a non-return valve into a high pressure manifold.

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In practice the low pressure manifold will be connected to a low pressure supply pipe or feed pipe which will carry an adequate supply of working fluid to meet the needs of the pump(s). Similarly, the high pressure manifold will in turn be connected to a high pressure supply or transmission pipe and it may also interface 5 with an accumulator capable of absorbing fluctuations or pulses in the output caused by the use of piston or plunger pumps (the use of an accumulator may not be necessary when rotary positive displacement pumps such as gear pumps are used, as their output tends to be relatively smooth and free from pressure pulses).

10 The low pressure and the high pressure transmission pipes will be led through the support structure of the tidal turbine to a convenient location for placing a water motor or Pelton Turbine connected to a generator. The water motor or Pelton turbine will generally be driven by one or more jets of water impinging on a specially shaped runner; such devices are prior art and well known.

15 In the application of this invention to the tidal turbine concept we have previously described in our British Patents GB 2311566 and 2348250, the turbines are mounted on a mono-piled support structure that is installed in a socket drilled into the seabed or river bed and the upper end of said support mono-pile emerges above the surface of the flowing water current. When the concept of the invention is applied to this method for extracting kinetic energy from currents then the low 20 pressure and the high pressure transmission pipes will carry the flow to and from a water motor or Pelton Turbine driving a generator that is preferably located above the surface of the water in a weather proof housing on the top of the supporting mono-pile.

25 Conveniently, however the housing for the water motor or Pelton Turbine may in some cases be a water tight casing located under the surface in the case of turbine/control systems that do not penetrate the water surface.

Preferably, the water used in the system of the present invention is filtered so as to be free of significant quantities of suspended solids or debris which would otherwise prematurely wear the components of both the pumps and the water motor or Pelton Turbine.

5 Conveniently, also the water may also be dosed with a non-harmful (bio-degradable or bio-friendly) lubricant to help minimise wear and tear of seals and moving components in the system.

Preferably any makeup water required for the proposals of the present invention is drawn through a strainer and then a filter by an ancillary pump and fed into a header tank or storage tank; it being arranged that said pump only functions at times when a float or other level indicating transducer signals that the water level in the header or storage tank has fallen below some predetermined level and ceases to function when the header or storage tank is replenished to another higher predetermined level.

10 15 In particular the aforementioned storage or header tank is linked to the water return feed pipe which collects water from below the Pelton Turbine or water motor and returns it to the feed manifold of the low speed, high-pressure pump (or pumps).

BRIEF DESCRIPTION OF THE DRAWINGS

20 For a better understanding of the invention and to show how to carry the same into effect reference will now be made to the accompanying drawings in which:-

Figure 1 is a schematic front view of a water current turbine system as disclosed in our co-pending British Patent Application No 0327429,7 the Figures illustrating the installation when the turbines and associated rotors are submerged in water;

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Figure 2 is a schematic side view of the installation of Figure 1,

Figure 3 is a section on the line A-A of Figure 1,

Figure 4 is a section on the line B-B of Figure 1;

5 Figures 5 and 6 schematically illustrate the installation of Figures 1 to 4, when the turbines and associated rotors are raised to a location above seal level; and

Figure 7 is a schematic diagram schematically illustrating an embodiment of the present invention when used with an installation illustrated in Figures 1 to 6.

Referring now to Figures 1 to 4, these Figures illustrate a support column 1 for mounting a turbine assembly 2. The lower end of the column 1 is embedded in a river/seabed SB and is of such length that the upper end thereof is above water level WL. In the Figures 1 to 4 the turbine assembly is shown as comprising two turbine units 3 mounted one to each of the extremities of a horizontal wing-like support structure 4 including an internal spar (not separately shown) within an outer housing (not separately shown) streamlined in cross section. The structure 4 is carried by a collar like sleeve 5 longitudinally displaceable lengthways of the support column 1. The profiling of the structure 4 is such as to assist in reducing as far as possible the creation of undesirable water drag effects in water passing over the surface of the structure 4 and to minimise the water wake at the trailing edge of the structure. In practice the direction of water flow across the structure 10 can be tidal and thus reversible in direction across the structure 4. This requires that the profiling of the structure has also to be bi-directional so that it is 15 operationally efficient in both directions of water flows.

Each turbine unit 3 includes a rotor 6 carried by a rotor shaft (not separately shown) that forms part of a very schematically indicated rotor rotation 20 transmission system 7 including a drive-train (not shown). The drive-train is 25

essentially a mechanism for applying the rotor shaft power for some useful purpose i.e., to generate electricity and may consist of a gearbox and generator, a direct drive generator or an hydraulic pump driving a generator via a hydraulic motor.

5 The sleeve 5 fits sufficiently closely to the upper section 8 of the column 1 as to be selectively displaceable length ways of the column as will be considered hereinafter. The upper section 8 of the column is also streamlined by being of an overall oval/elliptical profile so as to minimise the water flow wake and to maximise the resistance to bending in the directions of water flow normal to the

10 rotor plane.

In the embodiment of the system illustrated, the upper section 8 of the column comprises as shown in Figure 3 two face to face twin "D" profiled part columns 9 and 10 with a gap 11 between them through which the main spar of the support structure 4 can fit.

15 The sleeve 5, the structure 4, and the turbine units 3 are assembled together as a composite unit to form the assembly 2. This assembly is displaceable bodily lengthways of the upper section of the column by sliding the sleeve 5 up or down the upper section 8 of the column 1. The section 8 is of such length that the extent of possible displacement enables the assembly 2 to be positioned below water

20 level WL as is shown in Figures 1 and 2 and positioned above water level as is shown in Figures 5 and 6 so that the assembly 2 can be conveniently positioned with the rotors 6 in a stopped setting for the purposes of maintenance or other activities to be carried out upon the assembly.

25 In Figures 1 and 2 the assembly 2 is shown as being located with the axes of the turbine unit rotors 6 positioned approximately halfway between the water surface WL and the seabed SB.

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It will be noted from the Figure 4 section on the line B-B of Figure 2 that the lower or base section 13 of the column is of a solid circular cross-section. A housing 13 is provided at the upper end of the columns 9 and 10.

Whilst the above disclosed embodiment is considered advantageous and preferred by the Applicants but is not an essential element for applying this invention to a turbine installation..

Referring now to Figure 7, this Figure is a schematic diagram showing an embodiment of a system incorporating the concepts of the present invention and in particular illustrates how the system shown therein could be applied to an installation such as that shown in Figures 1 to 6.

Although Figure 7 shows the concepts of invention as applied to a turbine installation involving two rotors the concepts of the present invention could be applied to a single turbine installation or to installations involving more than two turbine

15 Referring now to Figure 7 in which a rotor 20 of a turbine installation (not shown in detail) is depicted in broken lines, the direction of rotor rotation being indicated by the part circular broken line 21.

20 The rotor 20 is arranged to rotate a shaft 22 mounting a cam drive 23 comprising in this example of an offset circular profile. The cam drive 23 is arranged to operate a pump assembly 24. As shown the assembly 24 incorporates four cylinders 25. The cylinders 25 are, in this example, symmetrically disposed around the cam drive 23 with their centre lines or longitudinal axes in the plane of the cam drive and radially directed with respect to the shaft 22.

25 The pistons or plungers 26 associated with the cylinders 25 are reciprocated within their associated cylinders 25 by roller type cam followers 27 forming part

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of the cam drive. In practice, return spring arrangements (not shown) can be provided inside each cylinder. The pump concept illustrated in Figure 7 can be regarded as being used purely by way of example. In practice, the pump unit/units used can be of any form suitable for the intended purpose. Other piston or plunger type pump unit configurations may be used. For example, gear, vane, lobe or other types of low speed, high pressure positive-displacement pumps could be used..

5 Inlet liquid i.e., inlet water for the pump assembly 24 is introduced into the cylinders 25 by way of a circular plenum or ring main 28. This water is introduced into each cylinder 25 by way of an associated non-return water inlet valve 29. That is each cylinder has a non-return water inlet valve 29.

10 It should be noted that the shape of this plenum or ring main 28 may vary to suit the form of the pump unit/pump installation being used and may not necessarily take the shape of a ring main 27 and four feed cylinders 25. For example, in some 15 installations involving a single entry point to a single pump a single feed pipe may well suffice.

20 The higher pressure water produced by the pumping action of the pump unit/units is expelled at this higher pressure from the pump cylinders 25 via non-return valves into a higher pressure water plenum or ring main 31. This plenum/ring main 30 interconnects to an optional surge-chamber or accumulator 32 capable of absorbing pulses in the liquid flow to produce a pressure wise smoothed output is fed by way of a higher pressure main 33.

Similarly feed water to the pumps arrives at the feed plenum 27 via a low pressure supply line 34.

25 In a multi-turbine installation as represented by Figure 7 the pump or pump assembly 24A associated with the other turbine rotors (one such pump unit being indicated at 24A, will incorporate plenums or ring mains 27A, 31A that are

hydraulically coupled to the above discussed outlet and inlet feed lines 33 and 34. In other words the inlet and outlet feed lines 34,33 can be common to all of the turbines of an installation.

5 The high pressure water main 33 transports the high pressure water emitted from the pumps 24 to a water motor or Pelton Turbine 35 that provides an efficient and well known mechanism for obtaining a high speed output from a jet (or in some cases several jets) of water. In this case the high pressure water main 33 terminates in one (or more) nozzles 36 which cause a high velocity jet (or jets) of water to impinge on the buckets or blades 37 of the water motor or Pelton Turbine 10 35. In many cases a multi-jet Pelton Turbine 35 may be used although the Figure for simplicity only shows the single jet option. The advantage of a multi-jet installation is that better part-load efficiency may be gained by using fewer jets when only a limited water supply is available.

15 The Figure 7 also shows the water motor or Pelton Turbine 35 driving an electrical generator 38 by way of pulleys and a belt transmission 39. This is primarily to clarify the principles being illustrated since in practice the generator 38 may in fact be direct coupled to the water motor or Pelton Turbine 35 (i.e. the shaft of the generator directly engaged with the shaft of the water motor), but equally some speed changing transmission such as a belt or belts may be used.

20 Since some of the water used as hydraulic fluid may leak from the circuit, especially from the high pressure pump or pumps (assemblies 24), any losses will need to be made up to ensure a continuous feed supply to the pump or pumps. In practice, it may well be desirable to allow some leakage since good clearances and less than perfect sealing will in all probability reduce the mechanical losses in the pump(s) and thereby improve the overall system efficiency. This arrangement in practice could also very probably make it easier to obtain long intervals before any pump seals need replacement.

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To accommodate any such leakage a make-up water supply 40 is provided in which a header tank 41 is mounted such that its water static level 42 is immediately below the water motor or Pelton Turbine 35 as illustrated, so that if the level in the return supply pipe 34 falls significantly below that in the header 5 tank, some make up water will flow from the header tank through a non-return valve 43 into the circuit near the top of the low pressure return supply pipe 33.

The header tank 41 is topped up when a level sensor 44 detects that the water level has fallen below some predetermined level; when this happens a small feed pump 45 is operated to draw water from the surroundings to the system through 10 an intake with a strainer and filter system 46 and to pump it through a makeup water feed pipe 47 into the header tank 41. When the water level in the header tank 41 exceeds a predetermined height, then the feed pump 45 is switched off. In most cases said feed pump will be electrically energised.

CLAIMS

- 1 A hydraulic transmission system for use with at least one water current driven turbine rotor, characterised in that a low speed high pressure pump or pumps (24,24A) is/are arranged to receive operational drive from the turbine rotor (6,20), and to deliver high pressure output to a high speed hydraulic motor or turbine (35) in such manner as to produce an increase of the shaft or motor speed accompanied by a corresponding reduction of torque required to drive electrical generators or other machinery (38) as compared with the shaft speed of the turbine rotor or rotors.
5
- 10 2. A hydraulic transmission system as in claim 1, and characterised in that a substantially closed hydraulic fluid circuit (27,31,33,34) is provided for said hydraulic fluid associated with the low speed high pressure pump or pumps (24,24A)
- 15 3 A hydraulic transmission system as claimed in claim 1 or 2, and characterised in that the hydraulic fluid for said high pressure pump or pumps is/are arranged to be recirculated from a high speed hydraulic motor or turbine (35) through a low pressure fluid return line or lines (34) to feed said low-speed high-pressure hydraulic pump or pumps (24,24A).
- 20 4. A hydraulic transmission system as in claim 1 2 or 3, and characterised in that the hydraulic fluid is water.
5. A hydraulic transmission system as claimed in claim 4, characterised in that the water required is originally be drawn from the water within which said turbine rotor or rotors (6) is/are is operationally located..

6. A hydraulic transmission system as claimed in claim 5, and characterised in that any loss of the water is arranged to be made up by drawing replacement water from the water within which said rotor or rotors (6,20) is/are operationally located.
- 5 7. A hydraulic transmission system as claimed in claim 6, and characterised in that such make up water is filtered to remove suspended solids or other undesirable pollutants before being stored in a header tank (41) situated in such a position that it can feed make-up water into the system to replace and water lost through leakage.
- 10 8. A hydraulic transmission system as in claim 4, 5,6 or 7, and characterised in that since the hydraulic fluid is water is drawn from the operating environment of the turbine or turbines , a relatively high degree of fluid leakage can be tolerated, thereby to allow the use of larger than would otherwise be acceptable clearances relative movement for seals, the arrangement being such that the operation of the pumps can be optimised for mechanical efficiency rather than 100% retention of hydraulic fluid.
- 15 9. A hydraulic transmission system as claimed in any one of the preceding claims, and characterised in that each said rotor (6,20) is associated with a plurality of separate pumps (24, 24A) arranged to be operationally driven from the associated rotor (20), and in that the hydraulic fluid inputs (29) to the pumps are connected to receive fluid from a low pressure fluid plenum (28) and the output sides of all said pumps are associated with a common high pressure plenum (31) connecting via high pressure fluid circuit (33) with the hydraulic motor or rotor (35) coupled to drive a generator or other machinery (38).
- 20 25 10. A hydraulic transmission system as claimed in claim 9, and characterised in that a surge or pressure balancing arrangement (32) is provided in the high pressure fluid circuit.

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11. A marine turbine installation incorporating a hydraulic transmission system as claimed in any one of the preceding claims.
12. A water drivable turbine installation including a hydraulic power transmission system for increasing the effective rotational speed of the turbine.

Fig. 1

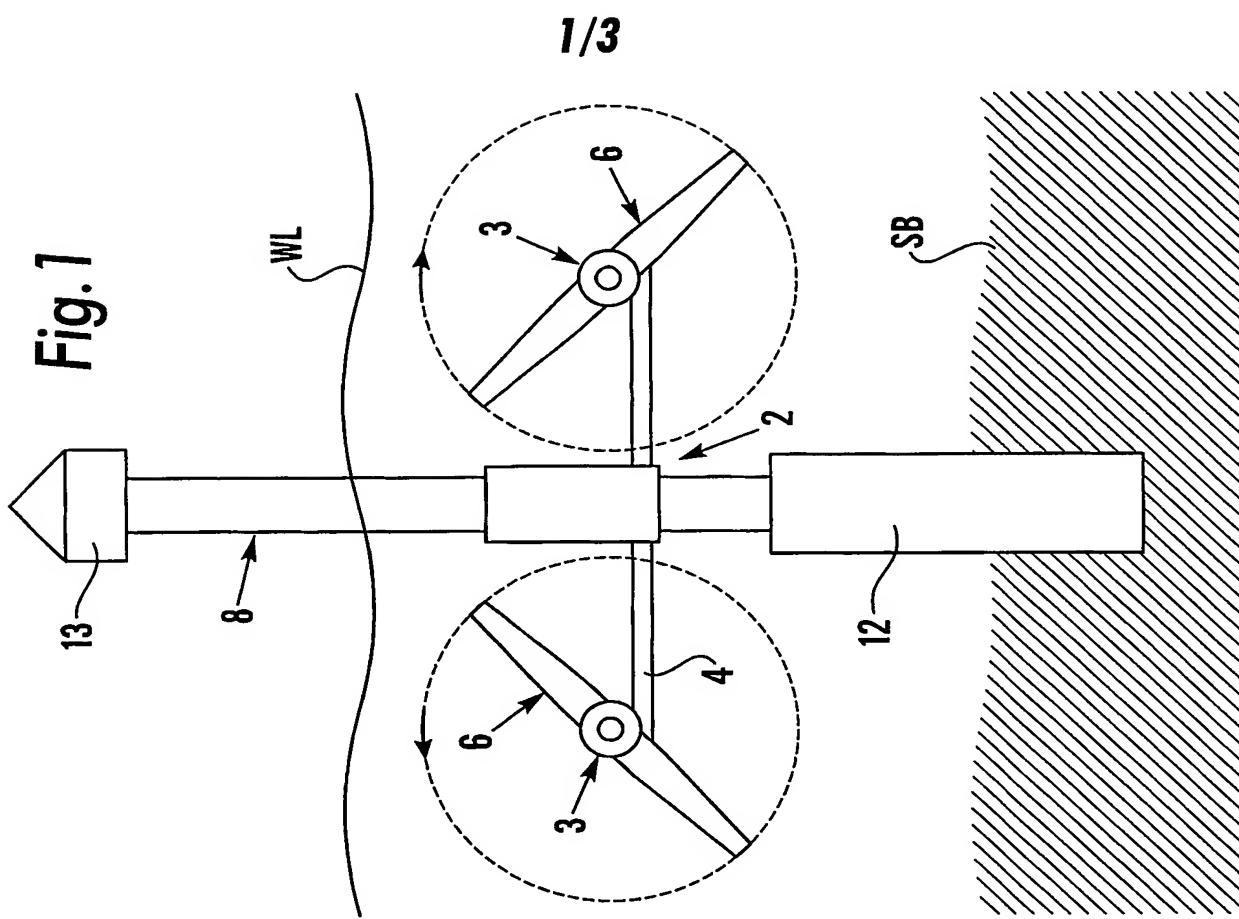


Fig. 2

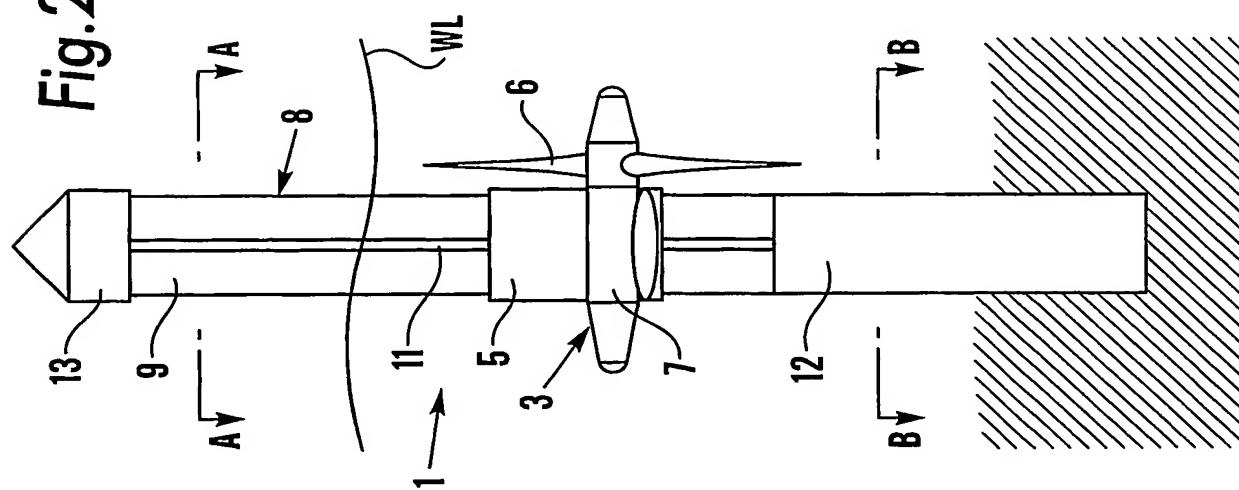


Fig. 3

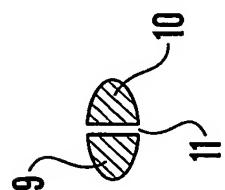
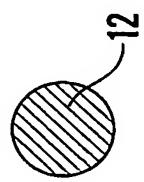


Fig. 4



2/3

Fig.6

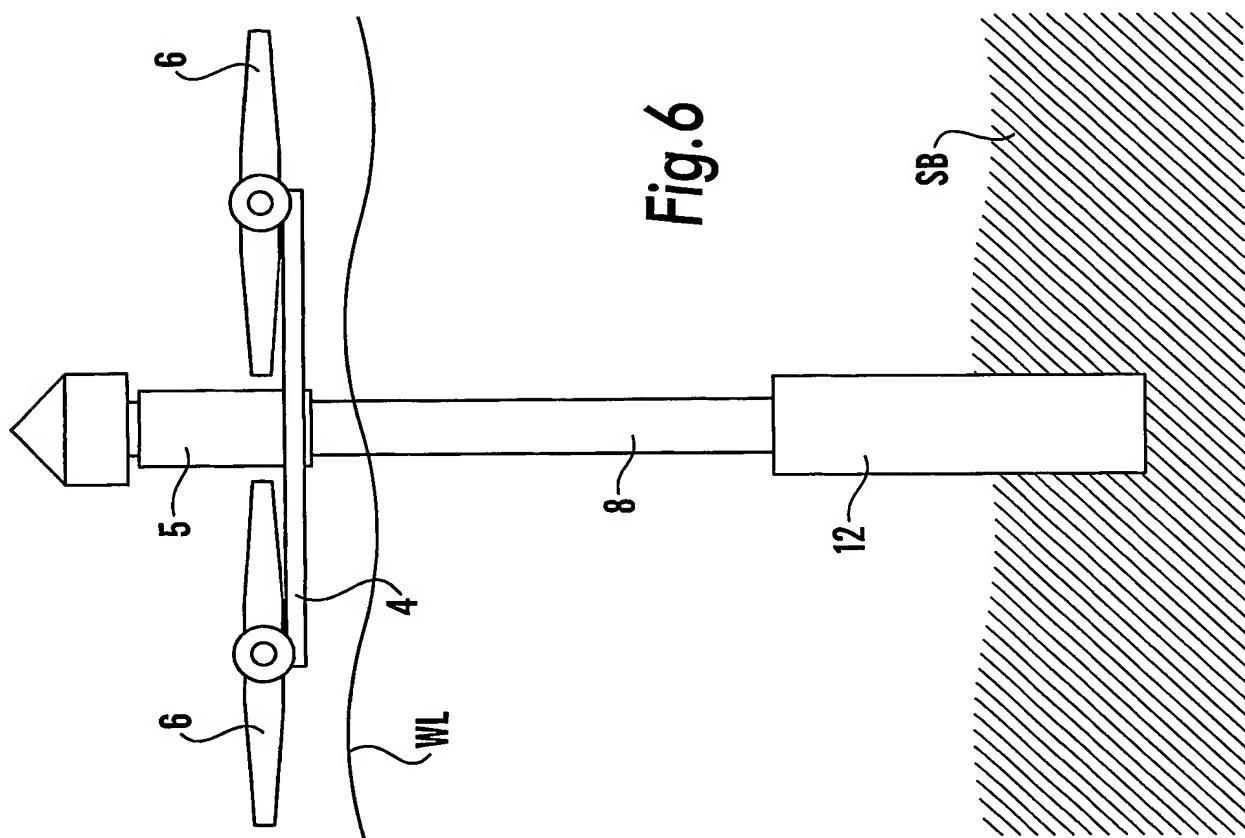
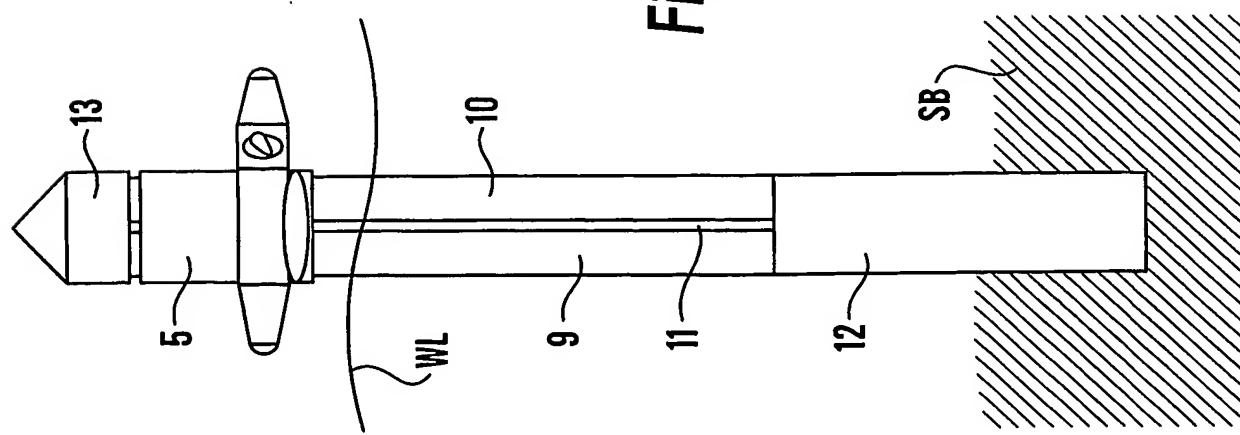


Fig.5



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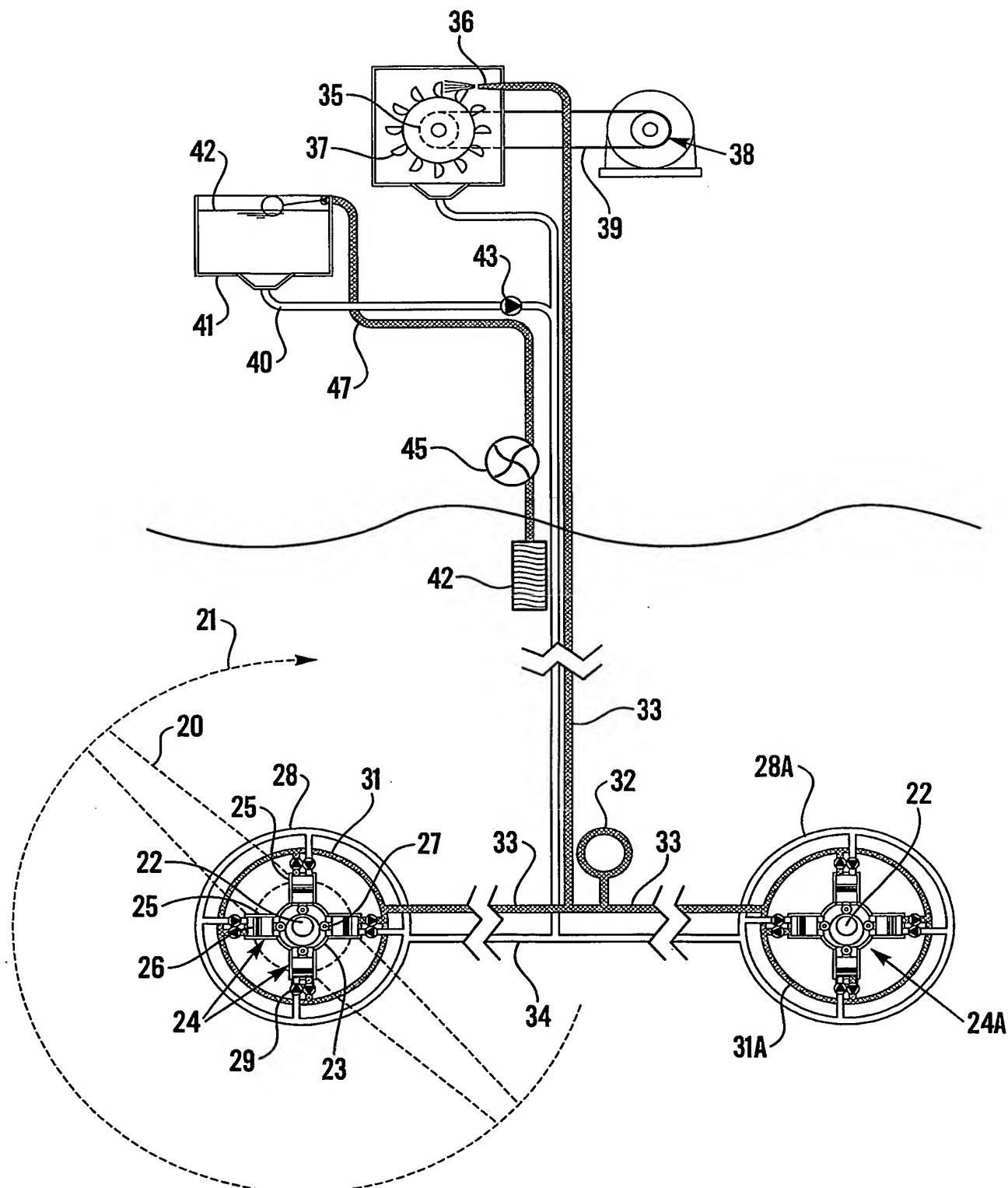


Fig. 7

INTERNATIONAL SEARCH REPORT

In onal Application No
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A. CLASSIFICATION OF SUBJECT MATTER

IPC 7 F03B17/06

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 F03B F03D F16H

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the International search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category ^o	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 4 383 182 A (BOWLEY WALLACE W) 10 May 1983 (1983-05-10) column 2, line 40-50 column 3, line 36-65 figures 2,3 ---	1-5
X	DE 195 30 253 A (WANZKE LOTHAR) 28 November 1996 (1996-11-28) column 3, line 10-29 column 4, line 25-38 figure 1 ---	1-5
A	GB 2 348 465 A (I T POWER LIMITED) 4 October 2000 (2000-10-04) the whole document --- -/-	1

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Patent family members are listed in annex.

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A	US 4 498 017 A (PARKINS WILLIAM E) 5 February 1985 (1985-02-05) the whole document -----	1

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